#### 1. 1 The different ways to construct effective medium

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# 1. 1 The different ways to construct effective medium

#### 1.1. 1.1 The problem and the direct solution

Consider two mass-in-mass elements, j = 1, 2, interacting through a soft spring of constant h. Each element has an outer mass  $M_j$  and an inner mass  $m_j$ , and a coupling spring of constant  $k_j$ . Calling the displacement of the outer mass  $U_j$  and  $u_j$  that of the inner mass, it comes that the system obeys the motion equation

$$\begin{bmatrix} k_1 + h & -k_1 & -h & 0 \\ -k_1 & k_1 & 0 & 0 \\ -h & 0 & k_2 + h & -k_2 \\ 0 & 0 & -k_2 & k_2 \end{bmatrix} \begin{bmatrix} U_1 \\ u_1 \\ U_2 \\ u_2 \end{bmatrix} = \omega^2 \begin{bmatrix} M_1 & & \\ & m_1 & \\ & & M_2 & \\ & & & m_2 \end{bmatrix} \begin{bmatrix} U_1 \\ u_1 \\ U_2 \\ u_2 \end{bmatrix}$$
(1)

For the unperturbated system, the equations are

$$\begin{aligned} k_1 & -k_1 & 0 & 0 \\ -k_1 & k_1 & 0 & 0 \\ 0 & 0 & k_2 & -k_2 \\ 0 & 0 & -k_2 & k_2 \end{aligned} \end{bmatrix} \begin{bmatrix} U_1 \\ u_1 \\ U_2 \\ u_2 \end{bmatrix} = \omega^2 \begin{bmatrix} M_1 & & \\ & m_1 & \\ & & M_2 \\ & & & m_2 \end{bmatrix} \begin{bmatrix} U_1 \\ u_1 \\ U_2 \\ u_2 \end{bmatrix}$$
(2)

The eigenvalues are

$$0, \ 0, \ \omega_1 = \frac{M_1 k_1 + k_1 m_1}{M_1 m_1}, \ \omega_2 = \frac{M_2 k_2 + k_2 m_2}{M_2 m_2} \tag{3}$$

and the corresponding eigenvectors are

$$e_{1} = [1, 1, 0, 0]^{T}$$

$$e_{2} = [0, 0, 1, 1]^{T}$$

$$e_{3} = [k_{1}, -k_{1}M_{1}/m_{1}, 0, 0]^{T}$$

$$e_{4} = [0, 0, k_{2}, -k_{2}M/m_{2}]^{T}$$
(4)

### 1.2. 1.2 First order degenerate pertubation theory

Assume the solution as the superposition of vector  $e_3$  and  $e_4$ 

$$u = xe_1 + ye_2 \tag{5}$$

Subsituting that into Eq. (1), we have

$$K(xe_1 + ye_2) = \omega M(xe_1 + ye_2) \tag{6}$$

Let product of  $e_1$  and  $e_2$  gives the equations of the coefficients x, y. The equations are

$$\begin{bmatrix} k_1^2 h + M_1^2 k_1 \omega_1^4 & -k_1 k_2 h \\ -k_1 k_2 h & k_2^2 h + M_2^2 k_2 \omega_2^4 \end{bmatrix} \begin{bmatrix} x \\ y \end{bmatrix} = \omega^2 \begin{bmatrix} M_1 k_1^2 \left(1 + \frac{M_1}{m_1}\right) & 0 \\ 0 & M_2 k_2^2 \left(1 + \frac{M_2}{m_2}\right) \end{bmatrix} \begin{bmatrix} x \\ y \end{bmatrix}$$
(7)

If  $k_1 = k_2, m_1 = m_2, M_1 = M_2$ , then  $\omega_1 = \omega_2$ . And Eq. (7) can be reduced to

$$\begin{bmatrix} k_1^2 h & -k_1 k_2 h \\ -k_1 k_2 h & k_2^2 h \end{bmatrix} \begin{bmatrix} x \\ y \end{bmatrix} = (\omega^2 - \omega_0^2) \begin{bmatrix} M_1 k_1^2 \left(1 + \frac{M_1}{m_1}\right) & 0 \\ 0 & M_2 k_2^2 \left(1 + \frac{M_2}{m_2}\right) \end{bmatrix} \begin{bmatrix} x \\ y \end{bmatrix}$$
(8)

Since this method is the first order degenerate perturbation theory, so the error exhibit  $O(h^2)$ .



Fig. 1. Frequency spectrum for  $k_1 = k_2 = 10$ , h = 1,  $m_1 = m_2 = M_1 = M_2 = 1$ . (a) Black lines are from exact solution Eq. (3), and red dash lines are from first degerate perturbation theory. (b) The error is proportional to  $h^2$ .



Fig. 2. Frequency spectrum for  $k_1 = 10, k_2 = 15, h = 1, m_1 = m_2 = M_1 = M_2 = 1$ . (a) Black lines are from exact solution Eq. (3), and red dash lines are from first degrate perturbation theory. (b) The error is proportional to  $h^2$ .

## 1.3. 1.3 The exact way

We transform the displacement vector to the new vector in the basis of eigenstates, or

$$\begin{bmatrix} U_1\\ u_1\\ U_2\\ u_2 \end{bmatrix} = \begin{bmatrix} 1 & 0 & k1 & 0\\ 1 & 0 & -k_1M_1/m_1 & 0\\ 0 & 1 & 0 & -k_2\\ 0 & 1 & 0 & -k_2M_2/m_2 \end{bmatrix} \begin{bmatrix} v_1\\ v_2\\ v_3\\ v_4 \end{bmatrix}$$
(9)  
or  $u = Uv$ .

Substituting Eq. (9) into Eq. (1), we have

$$U^T K U v = \omega^2 U^T M U v \tag{10}$$

or

$$\begin{bmatrix} h & -h & hk_1 & -hk_2 \\ -h & h & -hk_1 & hk_2 \\ hk_1 & -hk_1 & \frac{M_1^2k_1^3 + 2m_1M_1k_1^3}{m_1^2} + k_1^2(h+k_1) & -hk_1k_2 \\ -hk_2 & hk_2 & -hk_1k_2 & \frac{M_2^2k_2^3 + 2m_2M_2k_2^3}{m_2^2} + k_2^2(h+k_2) \end{bmatrix} \begin{bmatrix} v_1 \\ v_2 \\ v_3 \\ v_4 \end{bmatrix} = \omega^2 \begin{pmatrix} M_1 + m_1 & 0 & 0 & 0 \\ 0 & M_2 + m_2 & 0 & 0 \\ 0 & 0 & \frac{M_1k_1^2(M_1 + m_1)}{m_1} & 0 \\ 0 & 0 & 0 & \frac{M_2k_2^2(M_2 + m_2)}{m_2} \end{pmatrix} \begin{bmatrix} v_1 \\ v_2 \\ v_3 \\ v_4 \end{bmatrix}$$
(11)

Define  $H = K - \omega^2 M$ , and rewirte  $H = \begin{bmatrix} H_{11} & H_{12} \\ H_{21} & H_{22} \end{bmatrix}$  by 2 block matrices  $H_{ij}$ . Then the effective matrix of 2  $v_3, v_4$  is

$$H_{\rm eff} = H_{22} - H_{21} H_{11}^{-1} H_{12}.$$
 (12)

The spectrum can be obtained from the vanishing of determinant of Eq. (12), or





Fig. 3. Frequency spectrum for (a)  $k_1 = k_2 = 10$ , h = 1,  $m_1 = m_2 = M_1 = M_2 = 1$ . and (b)  $k_1 = 10$ ,  $k_2 = 15$ , h = 1,  $m_1 = m_2 = M_1 = M_2 = 1$ . Black lines are from Eq. (3) and red dash lines are from Eq. (13).

#### 1.4. 1.4 Schrieffer-Wolff transform

For simplicity, we set  $m_1=m_2=M_1=M_2=1$ , so Eq. (11) can be transformed into

$$\frac{1}{2} \begin{bmatrix} h & -h & h & -h \\ -h & h & -h & h \\ h & -h & h + 4k_1 & -h \\ -h & h & -h & h + 4k_2 \end{bmatrix} \begin{bmatrix} v_1 \\ v_2 \\ v_3 \\ v_4 \end{bmatrix} = \omega^2 \begin{bmatrix} v_1 \\ v_2 \\ v_3 \\ v_4 \end{bmatrix}$$
(14)

We separate Eq. (14) into the leading term and perturbative term, so we have

$$\frac{1}{2} \begin{bmatrix} h & -h & 0 & 0 \\ -h & h & 0 & 0 \\ 0 & 0 & h+4k_1 & -h \\ 0 & 0 & -h & h+4k_2 \end{bmatrix} \begin{bmatrix} v_1 \\ v_2 \\ v_3 \\ v_4 \end{bmatrix} + \begin{bmatrix} 0 & 0 & h & -h \\ 0 & 0 & -h & h \\ h & -h & 0 & 0 \\ -h & h & 0 & 0 \end{bmatrix} \begin{bmatrix} v_1 \\ v_2 \\ v_3 \\ v_4 \end{bmatrix} = \omega^2 \begin{bmatrix} v_1 \\ v_2 \\ v_3 \\ v_4 \end{bmatrix} \tag{15}$$
or  $Kv = \omega^2 v$ , where  $K = K_0 + K_1$ .

Now we want to do the Schrieffer-Wolff transform to separate the low frequency subspace to the high frequency subspace. We define the unitary transform  $v = \mathcal{U}w$  where  $\mathcal{U} = e^S$  and S is an anti-Hermitian matrix. After the unitary transform the stiffness matrix is transformed into

$$K' = e^S K e^{-S} \tag{16}$$

(13)

By using the Baker-Campbell-Haussdorf formula , we have

$$K' = K + [S, K] + \frac{1}{2}[S, [S, K]] + \dots$$
 (17)

or

$$K' = K_0 + K_1 + [S, K_0] + [S, K_1] - \frac{1}{2}[S, [S, K_0]] - \frac{1}{2}[S, [S, K_1]] + \dots$$
(18)

The Hamiltonian can be made diagonal to first order in  $K_1$  by choosing the generator S such that

$$[K_0, S] = K_1 (19)$$

This equation always has a definite solution under the assumption that  $hK_1$  is off-diagonal in the eigenbasis of  $K_0$ . Substituting this choice in the previous transformation yields:

$$K' = K_0 + \frac{1}{2}[S, K_1] + \mathcal{O}(h^3)$$
(20)

where K' is block diagonalized so we finish the separation of low frequency subspace and high frequency subspace. The error after SW transform are  $O(h^3)$  while it is  $O(h^2)$  for first degenerate perturbation theory. Actually, the SW transform is the second degenerate perturbation theory which is not commonly used. To get *S*, let's diagonalize  $K_0$  by producting a unitary transform matrix  $U_1$ 

$$\begin{bmatrix} \frac{1}{\sqrt{2}} & -\frac{1}{\sqrt{2}} & 0 & 0\\ \frac{1}{\sqrt{2}} & \frac{1}{\sqrt{2}} & 0 & 0\\ 0 & 0 & E_{11} & E_{12}\\ 0 & 0 & E_{21} & E_{22} \end{bmatrix}$$
(21)

where  $K_1E_i = d_iE_i$ . And  $K_0$  is diagonalized as

$$K_0' = \begin{bmatrix} 0 & 0 & 0 & 0 \\ 0 & h & 0 & 0 \\ 0 & 0 & d_1 & 0 \\ 0 & 0 & 0 & d_2 \end{bmatrix}$$
(22)

where

$$d_{1} = \frac{h}{2} + k_{1} + k_{2} - \frac{\sqrt{h^{2} + 4k_{1}^{2} - 8k_{1}k_{2} + 4k_{2}^{2}}}{2}$$

$$d_{2} = \frac{h}{2} + k_{1} + k_{2} + \frac{\sqrt{h^{2} + 4k_{1}^{2} - 8k_{1}k_{2} + 4k_{2}^{2}}}{2}.$$
(23)

And  $K_1$  are transformed into

$$K_1' = \mathcal{U}_1^T K_1 \mathcal{U}_1 \tag{24}$$

and it is

$$\begin{bmatrix} 0 & 0 & K'_{11} & K'_{12} \\ 0 & 0 & K'_{21} & K'_{22} \\ K'_{11} & K'_{12} & 0 & 0 \\ K'_{21} & K'_{22} & 0 & 0 \end{bmatrix}$$
(25)

And the solution of S is

$$S_{ij} = \begin{cases} \frac{K'_{ij}}{d_j - d_i} & i \neq j \\ 0 & i = j \end{cases}$$

$$(26)$$

and then from Eq. (20), we have

$$K'_{d} = K'_{0} + \frac{1}{2} [S, K'_{1}] + \mathcal{O}\left(h^{3}\right)$$
(27)

Rotating back give the stiffness matrix whose low frequency subspace and high frequency subspace are separated

$$K' = \mathcal{U}_1 K'_d \mathcal{U}_1^T \tag{28}$$



Fig. 4. Frequency spectrum for  $k_1 = k_2 = 10$ , h = 1,  $m_1 = m_2 = M_1 = M_2 = 1$ . (a) Black lines are from exact solution Eq. (3), and red dash lines are from SW transformation. (b) The error is proportional to  $h^3$ .



Fig. 5. Frequency spectrum for  $k_1 = 10, k_2 = 15, h = 1, m_1 = m_2 = M_1 = M_2 = 1$ . (a) Black lines are from exact solution Eq. (3), and red dash lines are from SW transformation. (b) The error is proportional to  $h^3$ .